

3.7. PROJEKTOVANJE ZAVARENIH KONSTRUKCIJA – STATIČKO OPTEREĆENJE

1. Vrste zavarenih spojeva

U tab. 3.2.1. su date osnovne vrste zavarenih spojeva koje se proračunavaju prema DIN 18 800/ Deo 1, sa njihovim oznakama (simbolima) prema DIN 1912/ Deo 5, a u tab. 3.2.2 proračunske debljine.

Tabela 3.2.1. Osnovne vrste zavarenih spojeva za proračun prema DIN 18 800

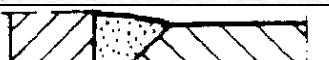
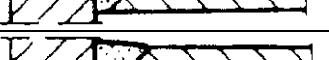
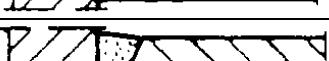
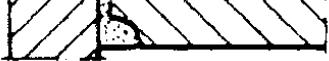
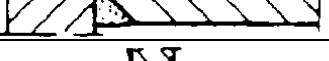
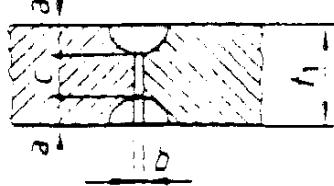
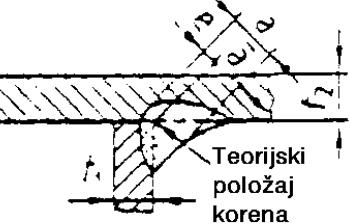
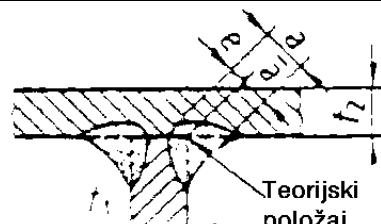
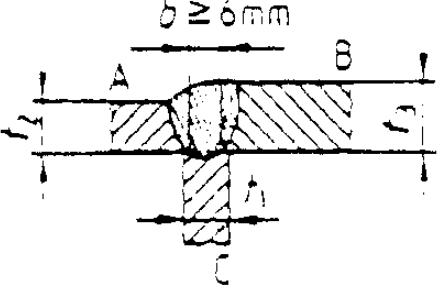
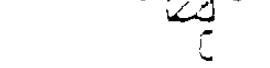
Redni broj	Vrsta spoja	Obeležavanje	Skica i simbol
1.	Sučeoni spoj, jednostrani ili obostrani	I	
2.		V	 \
3.		Y	 Y
4.		U	 U
5.		DV (X)	 X
6.		DY	 Y
7.		DU	 U
8.	Druge vrste spojeva, jednostrani ili dvostrani	HV	 \
9.		DHV	 K
10.		HU	 U
11.		DHU	 K
12.	Neprovareni spojevi	HY	 Y
13.		DHY	 Y
14.	Ugaoni spojevi	Ugaoni spoj	 \
15.		Dvostrani ugaoni spoj	 \
16.	T spoj sa tri elementa		

Tabela 3.2.2. Određivanje debljine zavarenih spojeva

Redni broj	Vrsta spoja	Skica	Proračunska debljina, a
1.	2.	3.	4.
1.	Sučeoni spoj		$a = t_1$
2.	DHV (K) šav		
Jednostrani ili dvostrani šav	HV šav	Pokrivni zavar sa druge strane	 $a = t_1$
		Provaren koren	
5.	HY šav ugaoni šav		
6.	HY šav		
Šavovi bez provara	DHY šav sa dvostranim ugaonim šavom		Debljina šava a je jednaka rastojanju od teorijskog položaja korena do površine šava
	DHY šav		

1.	2.		3.	4.
9.	Šavovi bez provara	DHY šav bez pripreme šava		Debljina šava a se određuje prema procesu zavarivanja. Zazor b zavisi od procesa. Za EPP proces $b=0$
10.	Ugaoni šavovi	Ugaoni šav		Debljina šava je jednaka izmerenoj visini upisanog ravnokrakog trougla do teorijskog položaja korena
11.		Dvostrani ugaoni šav		Teorijski položaj korena
12.		Ugaoni šav		$a = \bar{a} + e$ \bar{a} – odgovara debljini šava prema stavkama 10 i 11 e – određeno prema procesu zavarivanja (DIN 18 800/ Deo 7)
13.		Sa dubokim provarom		Teorijski položaj korena
14.	Ivični šav tri lima	Prenos sile od A na B		$a = t_2$ za $t_2 < t_3$
15.		Prenos sile od C na A i B		$a = b$

Za šavove pod r. br. 5. do 8. sa uglom žleba od 45° proračunska a veličina treba da se smanji za oko 2 mm ili da se utvrdi postupkom zavarivanja. Od toga se izuzimaju šavovi izvedeni u koritastom i horizontalnom položaju pri zavarivanju u zaštiti gasom.

U tab. 3.2.3. su date proračunske dužine ugaonih spojeva u direktnim vezama.

Tabela 3.2.3. Dužine ugaonih spojeva u direktnim vezama

Redni broj	Vrsta zavarenog spoja	Skica	Proračunska dužina šava $\sum l$
1.	Ugaoni šav ivica		$\sum l = 2l_1$
2.	Ugaoni šav ivica i čela		$\sum l = b + 2l_1$
3.	Ugaoni šav po obimu – težište bliže dužem šavu		$\sum l = l_1 + l_2 + 2b$
4.	Ugaoni šav po obimu – težište bliže kraćem šavu		$\sum l = 2l_1 + 2b$
5.	Ugaoni ili HV šav prorezanog ugaonika		$\sum l = 2l_1$

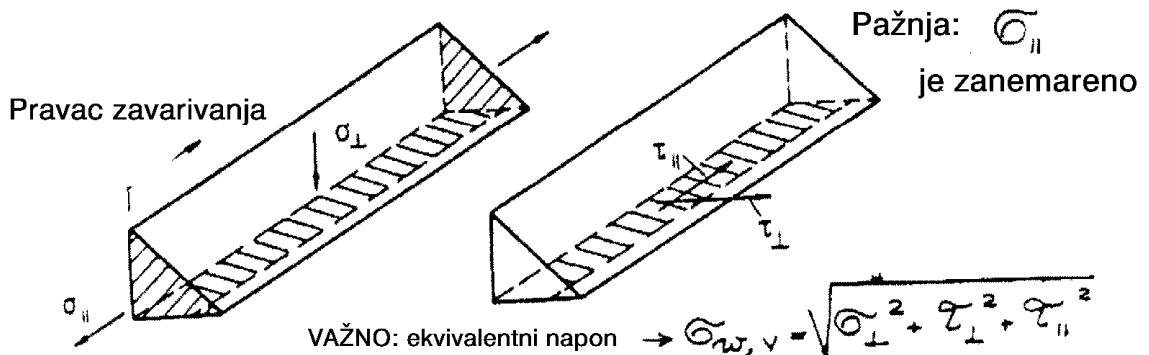
2. Proračun čvrstoće zavarenih spojeva

Osnove proračuna su iznete prema DIN 18 800/ Deo 1.

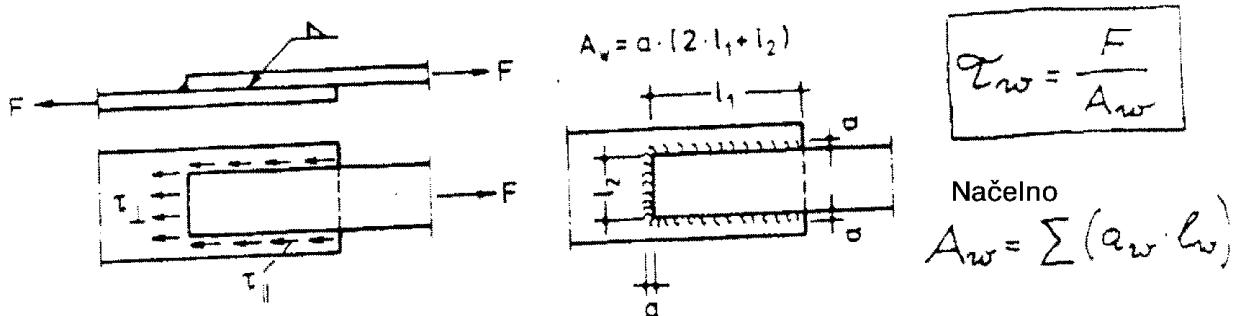
- Zavareni spojevi prenose:**

- normalno zatezanje (jedinični napon) $\rightarrow \sigma (\sigma_{\perp} \text{ i } \sigma_{\parallel})$

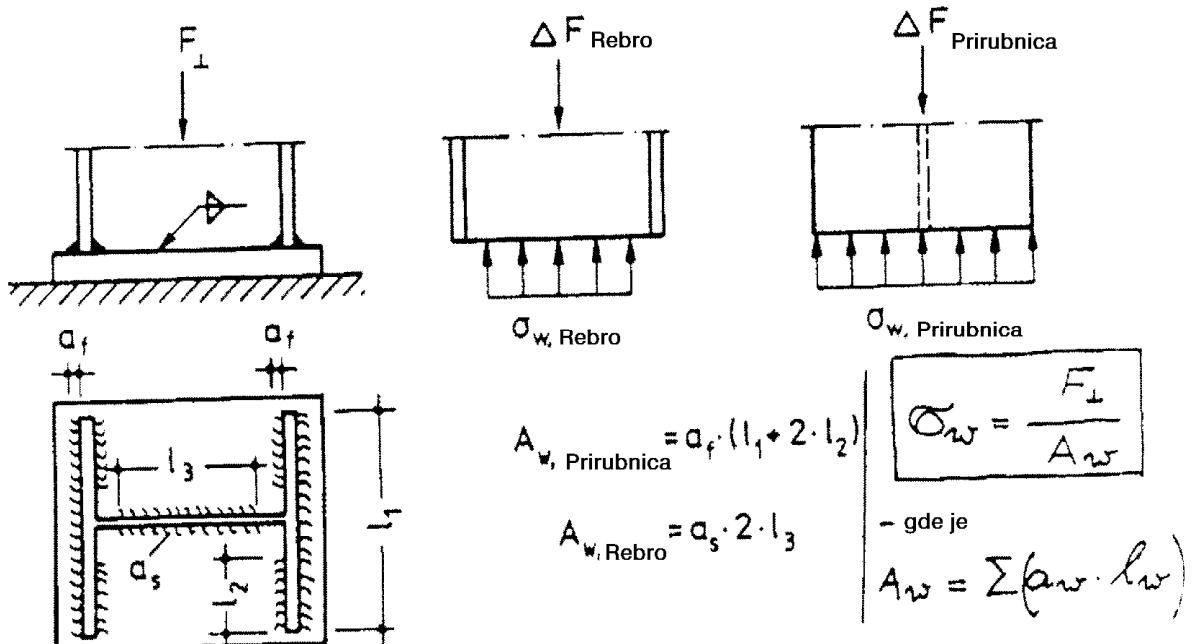
- napon smicanja $\rightarrow \tau (\tau_{\perp} \text{ i } \tau_{\parallel})$



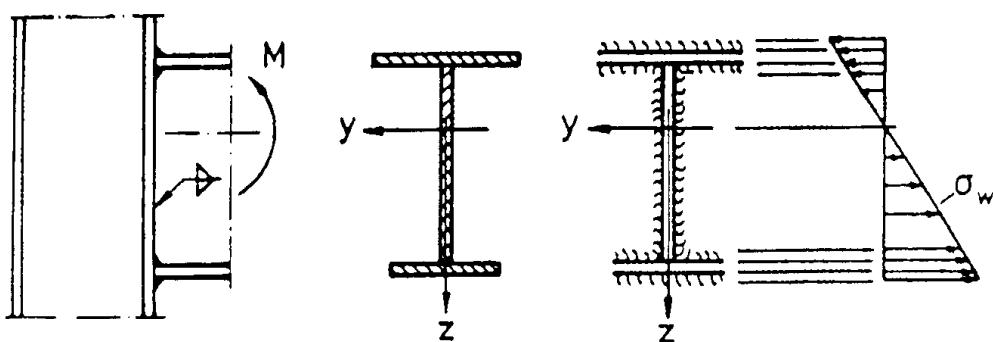
- Prenos sила smicanja



- Prenos sила upravnih na spoj



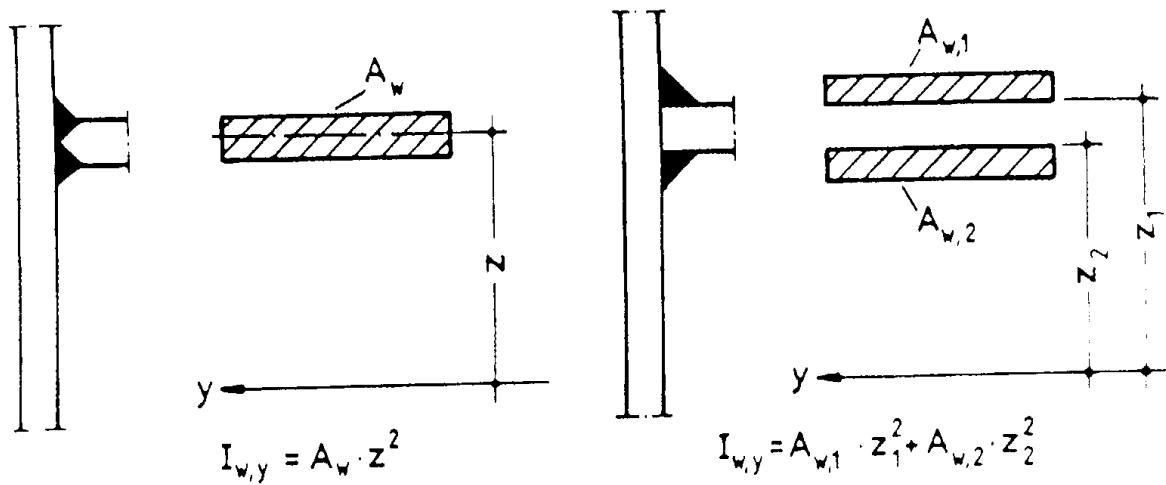
• Prenos momenata savijanja



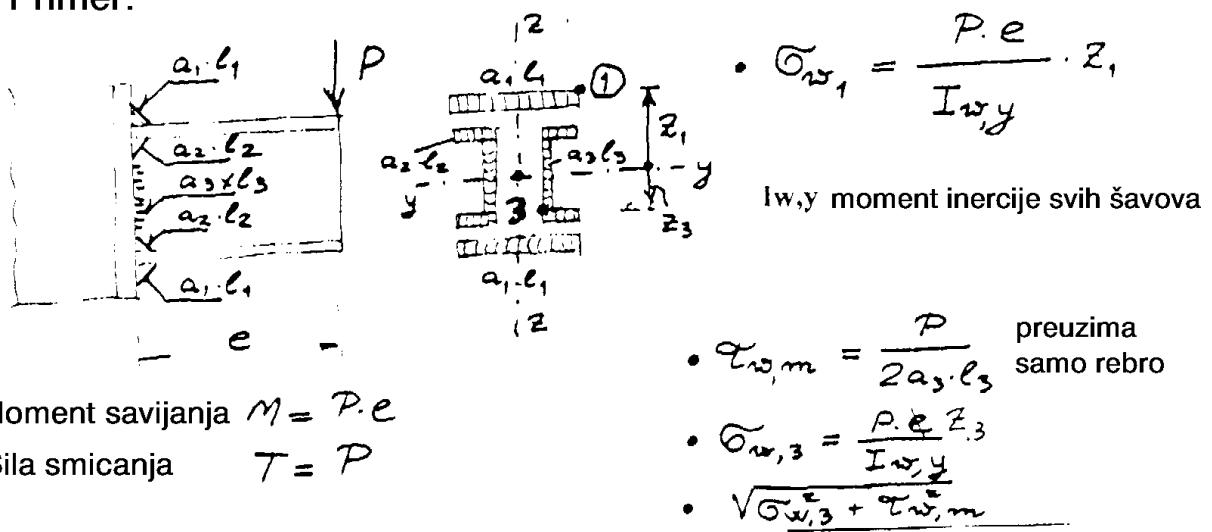
$$\sigma_w = \frac{M_y}{I_{w,y}} \cdot z$$

Glavne ose napona se moraju poklapati sa glavnim osama šava

Proračun momenta inercije $I_{w,y}$ (drugog momenta površine) za šav prirubnice:

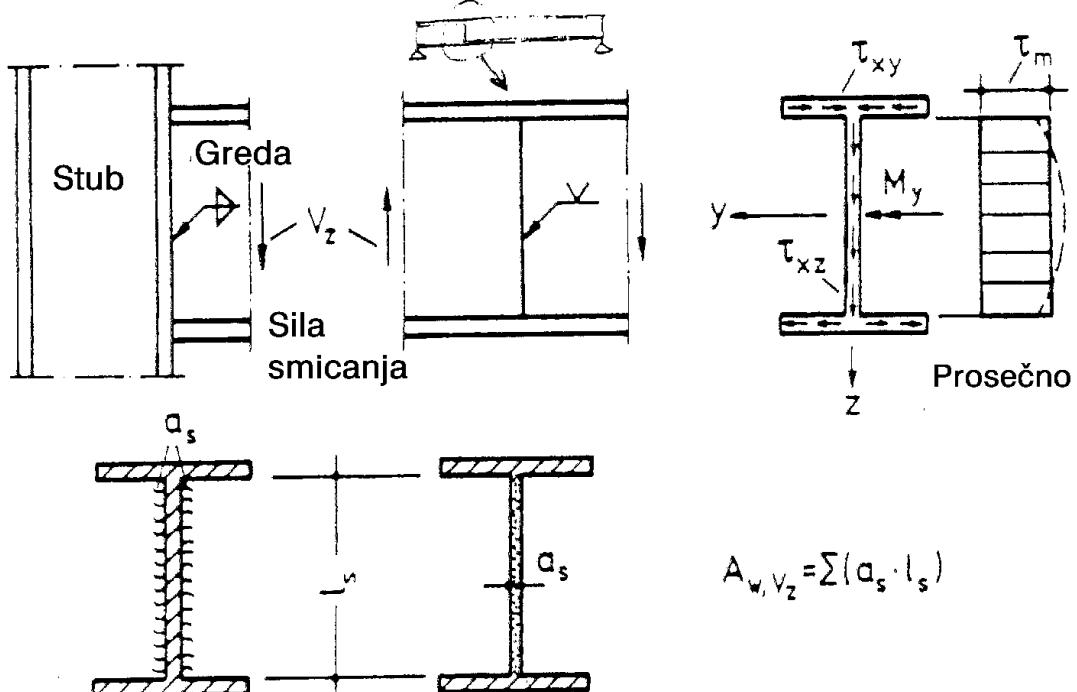


Primer:



- Prenos sila smicanja

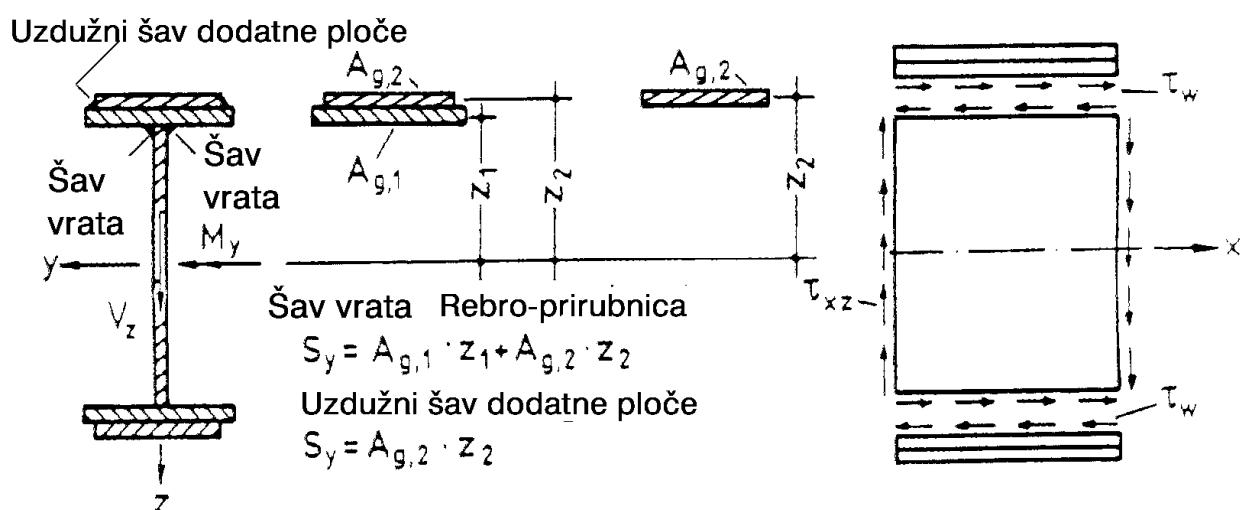
a.) Spoj rebra; spoj stub-greda



Zatezanje se prenosi samo preko rebra

$$\tau_{w,m} = V_z / A_{w,V_z}$$

Uzdužni spoj rebra, spoj rebro-prirubnica



$$\tau_w = \frac{V_z \cdot S_y}{(\sum a_w) \cdot I_y}$$

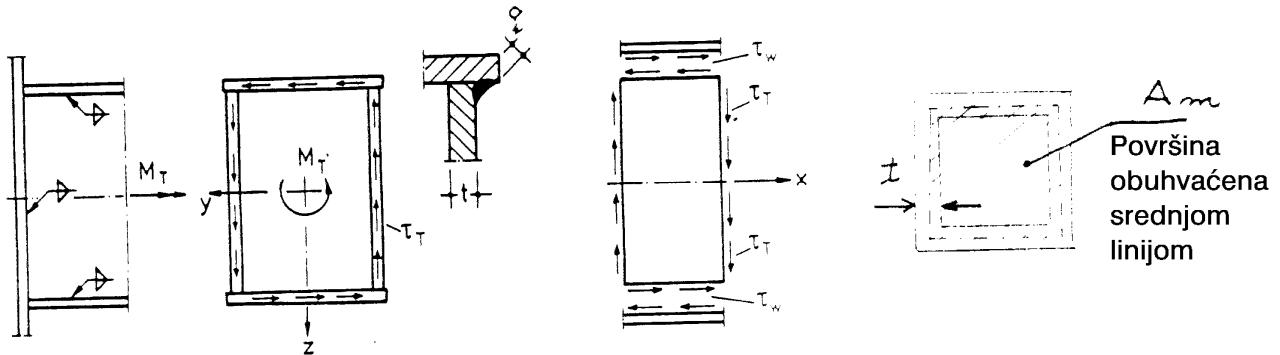
Jednačina
Mor - Žuravski

• S_y Statički moment za neutralnu osu dela preseka izloženog smicanju
 I_y Moment inercije preseka

• Prenos momenta torzije

Moment torzije uvodi napon smicanja u zatvorenim presecima i njihovim spojevima (τ_t – Sen Venan). Odnos napona smicanja u šavu zavarenog spoja i u osnovnom preseku je $\tau_w / \tau_T = t/a_w$. Napon smicanja se računa po formuli Bretta:

$$\tau_T = \frac{M_T}{2A_m t}$$



Analiza spoja prema DIN 18 800

Osnovna formula maksimalnog graničnog stanja za projektno stanje (indeks "d"):

$$S_d/R_d \leq 1$$

Za zavareni spoj važi formula za uporedni napon (indeks "v")

$$\sigma_{w,v} \leq \sigma_{w,R,d}$$

Konvencionalna vrednost normalnog napona je

$$\sigma_{w,v} = \sqrt{\sigma_\zeta^2 + \tau_\zeta^2 + \tau^2}$$

Projektni napon za šav čelika napona tečenja $f_{y,k}$:

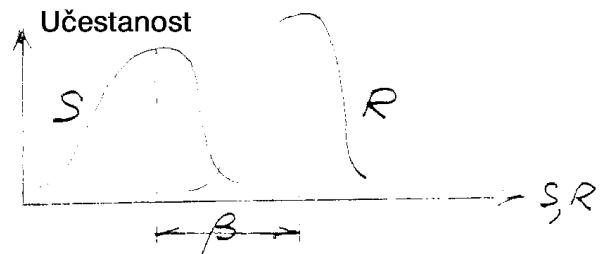
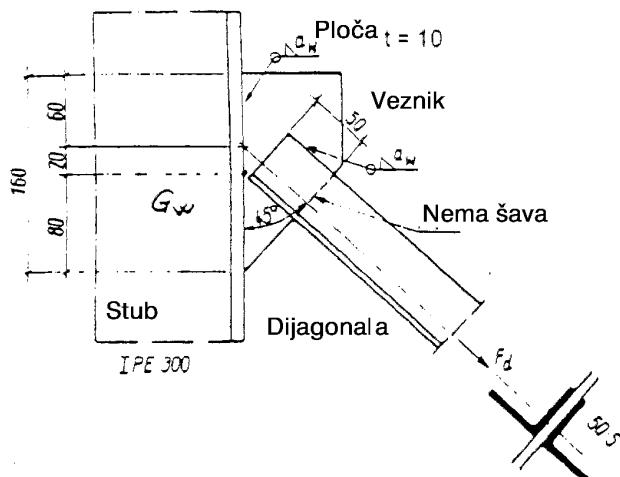
$$\sigma_{w,R,d} = \alpha_w f_{y,k} / \gamma_M$$

$f_{y,k}$ za čelik St 37 je 240 MPa, za St 52 je 360 MPa.

Koeficijent α_w koji zavisi od vrste spoja, kvaliteta šava i kvaliteta materijala, iznosi 0,80 do 1,00. Veličina γ_M je parcijalni stepen sigurnosti materijala.

Primer

Proveriti zavareni spoj između ugaonika i veznika i veznika i stuba na slici.



Podaci:

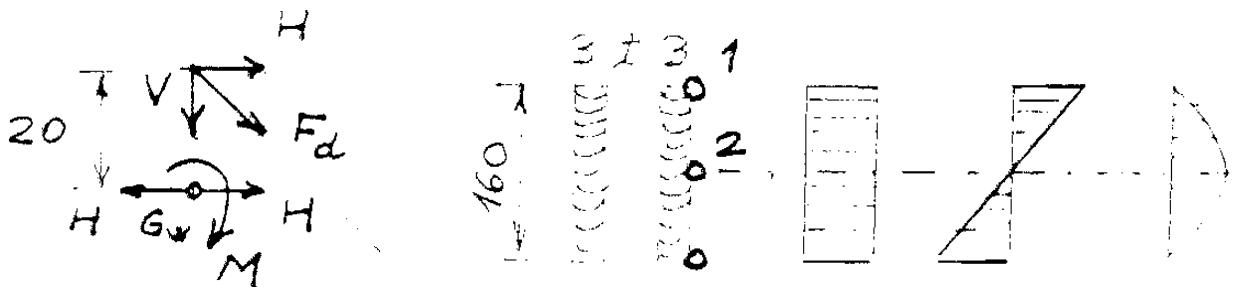
St37; $\gamma_M = 1,1$; $a_w = 3 \text{ mm}$; $F_d = 150 \text{ kN}$

Dijagonalni šav – veznik (ploča):

- dužina šava $\Sigma l = b + 2l_I = 50 + 2 \times 50 = 150 \text{ mm}$
 - površina šava $A_w = 2 \times 15 \times 0,3 = 9 \text{ cm}^2$
 - napon smicanja $\tau_{||} = 150/9 = 15,7 \text{ kN/cm}^2$
 - ekvivalentni napon $\sigma_{w,v} = \sqrt{16,7^2} = 16,7 \text{ kN/cm}^2$
 - koeficijent $\alpha_w = 0,95$
 - napon tečenja $f_y = 24 \text{ kN/cm}^2$
 - projektni napon $\sigma_{w,R,d} = 0,95 \times 24 / 1,1 = 20,7 \text{ kN/cm}^2$
 - provera prema DIN 18 800
- $\sigma_{w,v} / \sigma_{w,R,d} = 16,7 / 20,7 < 1$

Spoj veznik – stub:

Prvi korak: sve sile treba redukovati na težište šava G_w , čija je ekscentričnost 20 mm, kako je prikazano na slici dole levo.



- Horizontalna i vertikalna komponenta sile F_d

$$H = F_d \times \sin \alpha = 150 \times \sqrt{2}/2 = 106,1 \text{ kN}$$

$$V = F_d \times \cos \alpha = 150 \times \sqrt{2}/2 = 106,1 \text{ kN}$$

- Moment $M = 106,1 \times 2 = 212,2 \text{ kNm}$

- Karakteristike šava: $l_w = 16 \text{ cm}$; $A_w = 2 \times 0,3 \times 16 = 9,6 \text{ cm}^2$; $I_w = 2 \times (0,3 \times 16^3)/12 = 205 \text{ cm}^4$

Provera gornje tačke (1) šava (slika gore desno):

$$\sigma_{\perp} = 106,1/9,6 + (212,2/205) \times 8 = 19,4 \text{ kN/cm}^2 = \sigma_{w,v}; \text{ sa koeficijentom } \alpha_w = 0,95 \text{ i projektnim naponom } \sigma_{w,R,d} = 20,7 \text{ kN/cm}^2 \text{ to daje } \sigma_{w,v} / \sigma_{w,R,d} = 19,4/20,7 < 1$$

Provera sredine šava (tačka 2)

- Statički moment $S = 2 \times 0,3 \times 8 \times 8/2 = 19,2 \text{ cm}^2$
- Napon smicanja (Mor-Žuravski): $t = (106,1 \times 19,2)/2 \times 0,3 \times 205 = 16,6 \text{ kN/cm}^2$
- Normalni napon $\sigma_{\perp} = 106,1/9,6 = 11,1 \text{ kN/cm}^2$
- Ekvivalentni napon $\sigma_{w,v} = \sqrt{16,6^2 + 11,1^2} = 20,0 \text{ kN/cm}^2$

$$\sigma_{w,v} / 20,7 < 1$$

Zaključak je da je otpornost šava zadovoljavajuća.

Napomene:

1. Izbor debljine ugaonog šava za $t \geq 3 \text{ mm}$ je uslovljen sa

$$2 \text{ mm} \leq a_w \leq 0,7t, \text{ odnosno } a_w = \sqrt{t_{max}} - 0,5, \text{ u mm.}$$

2. Zavareni spoj je pod uglom u odnosu na veznik.

Prvo rešenje, preporučeno prema DIN

18 800 (prosto za izvođenje)

$$A_{w1} = A_{w2} = a_w \times l$$

Druge rešenje

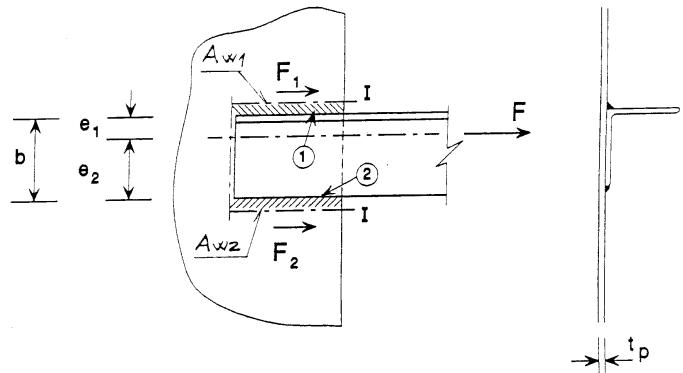
$$A_{w1} \neq A_{w2} (A_{w1} > A_{w2})$$

$$A_{w1} = a_{w1} \times l_1$$

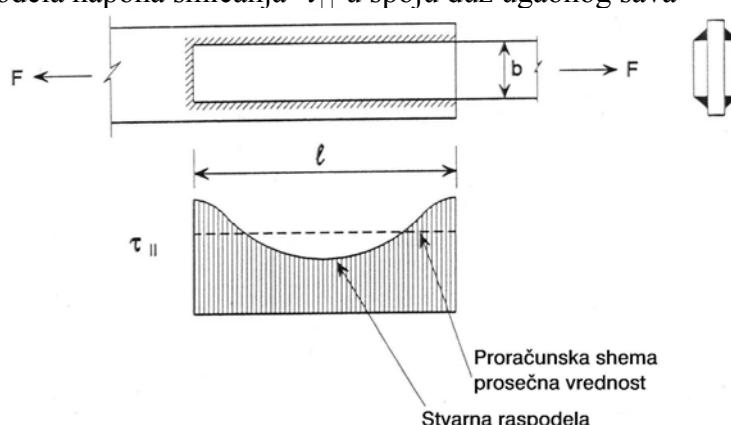
$$A_{w2} = a_{w2} \times l_2$$

$$A_w = A_{w1} + A_{w2} = F / \sigma_{w,R,d}$$

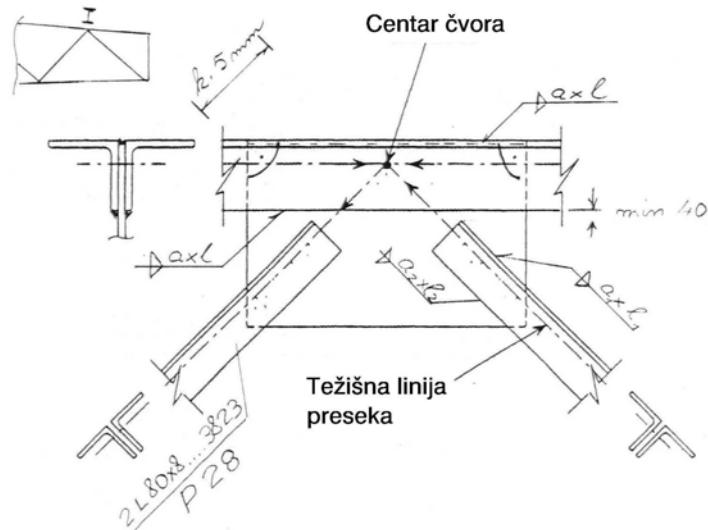
$$A_{w1} \times e_1 = A_{w2} \times e_2$$



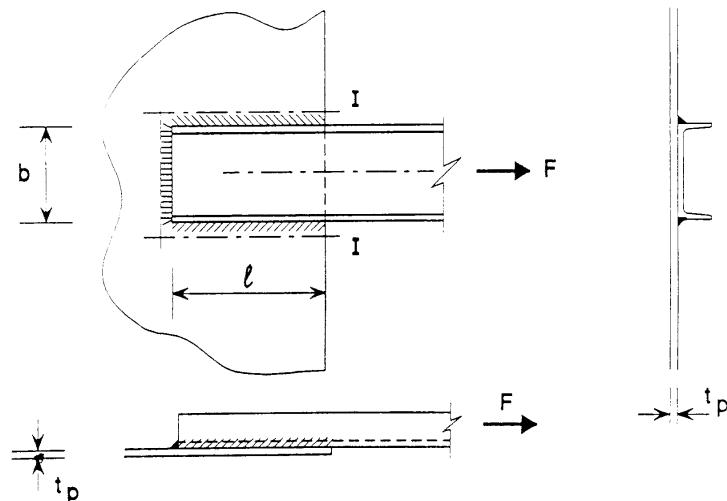
3. Neravnomerna raspodela napona smicanja $\tau_{||}$ u spoju duž ugaonog šava



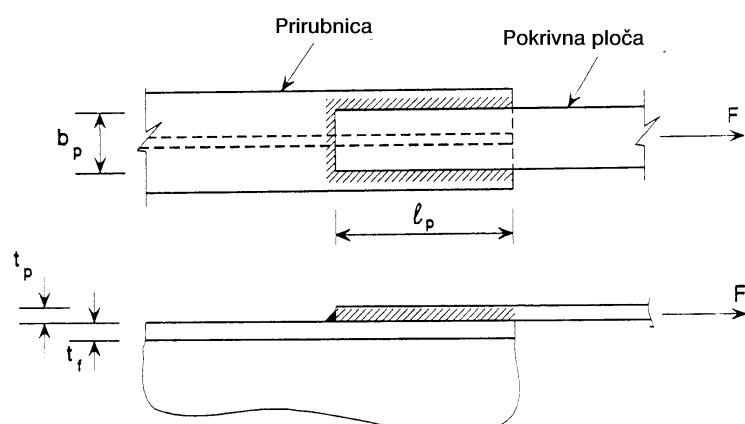
4. Spoj između krovne rešetke i veznika



5. Spoj tunelske sekcije sa veznikom

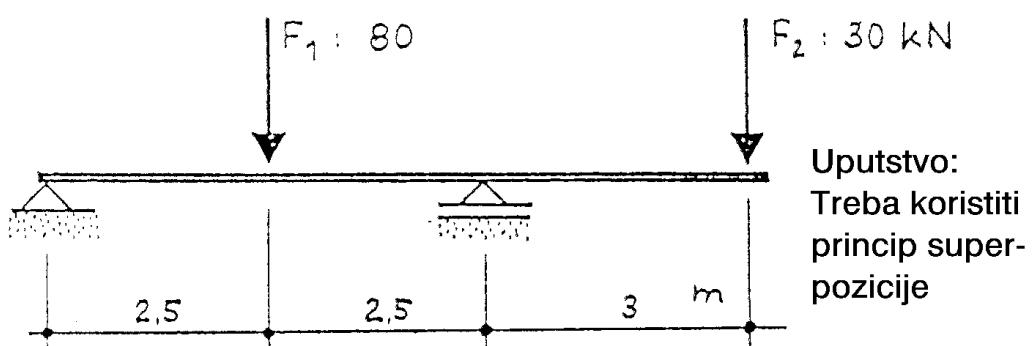
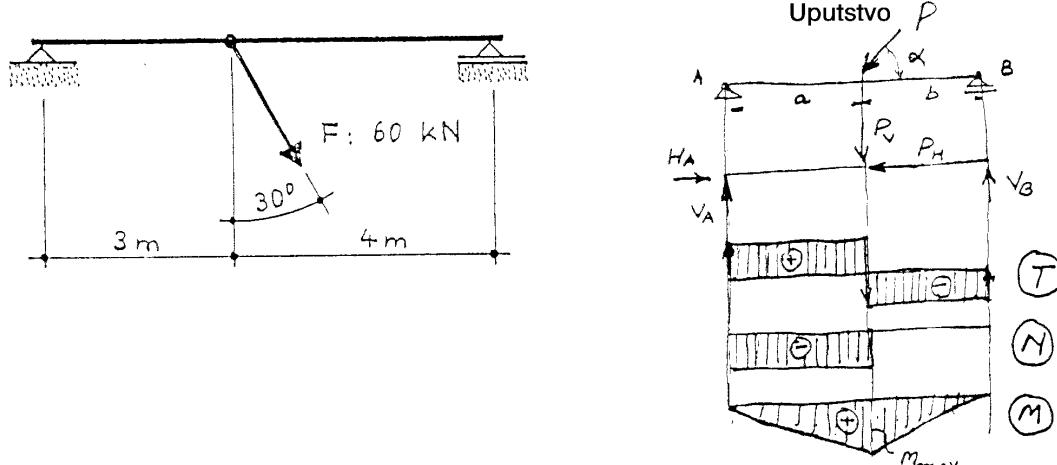
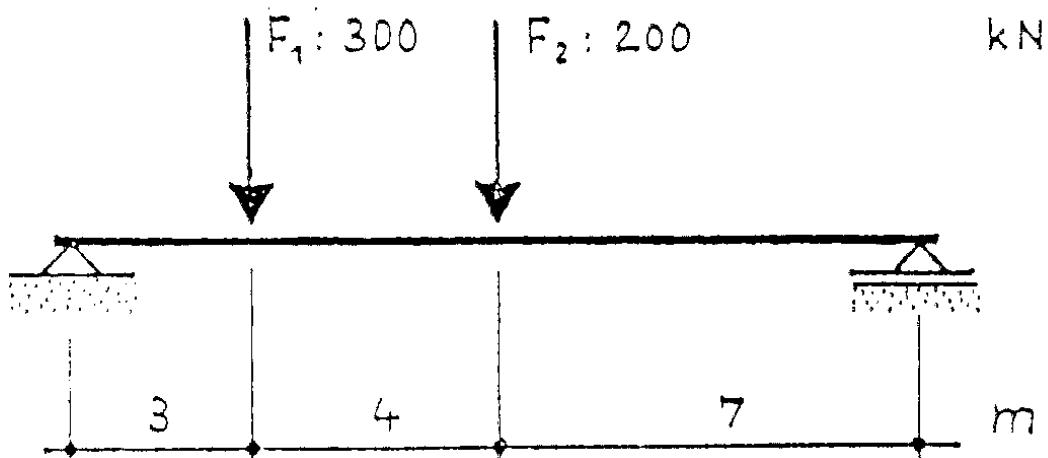
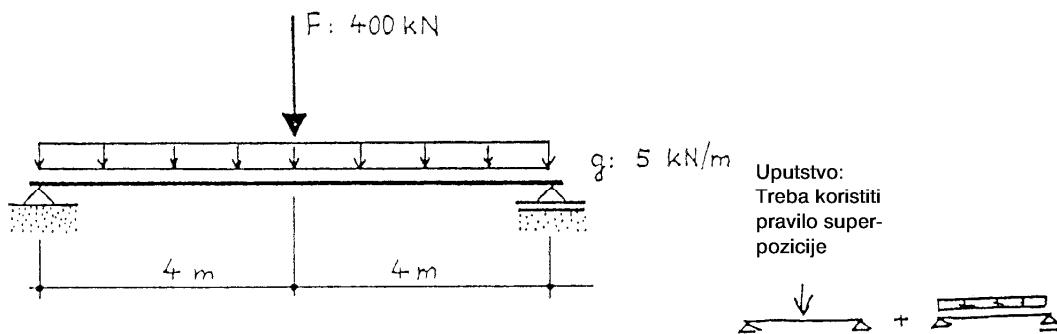


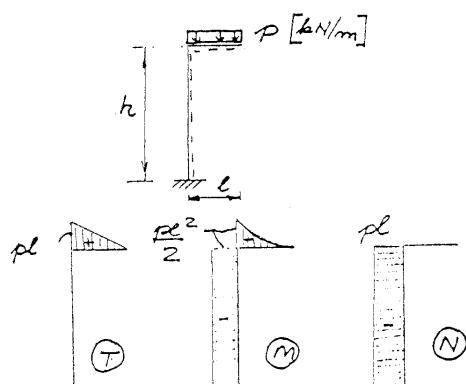
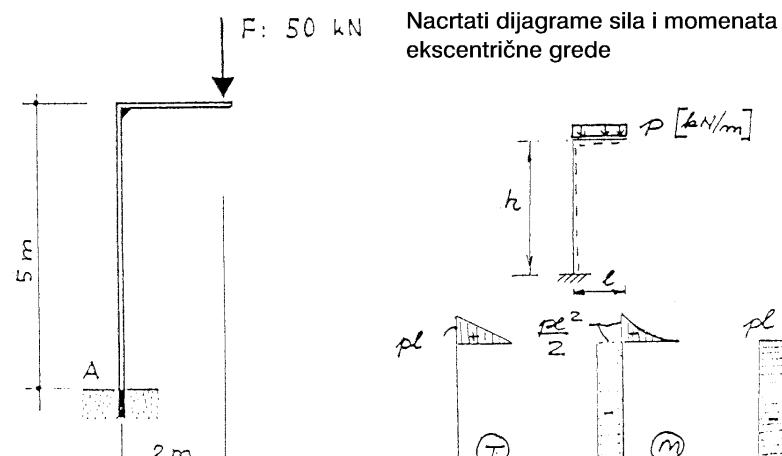
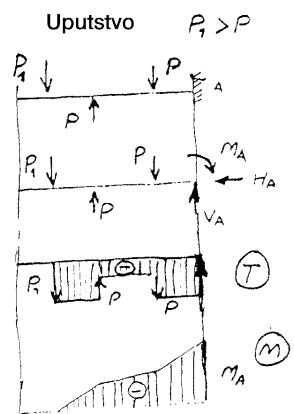
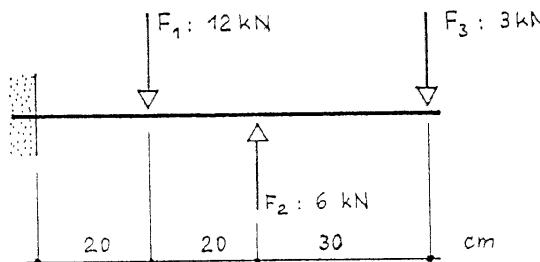
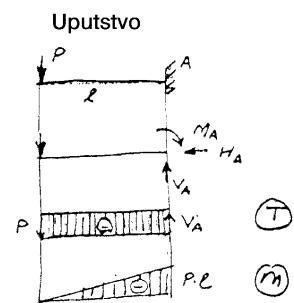
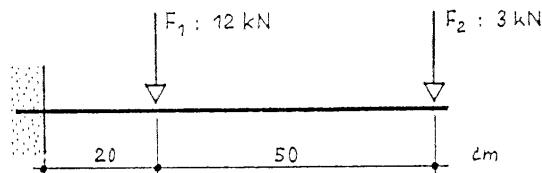
6. Zavarena pokrивna ploča



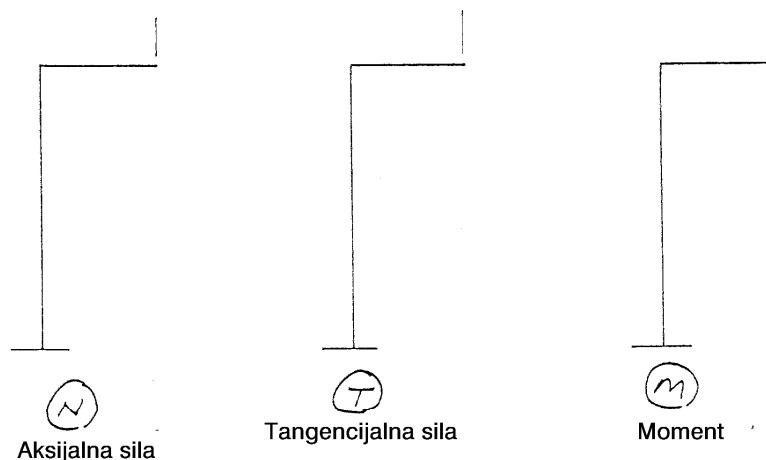
Zadaci za vežbu

Za naredne konstrukcije odrediti dijagrame sila i momenata.





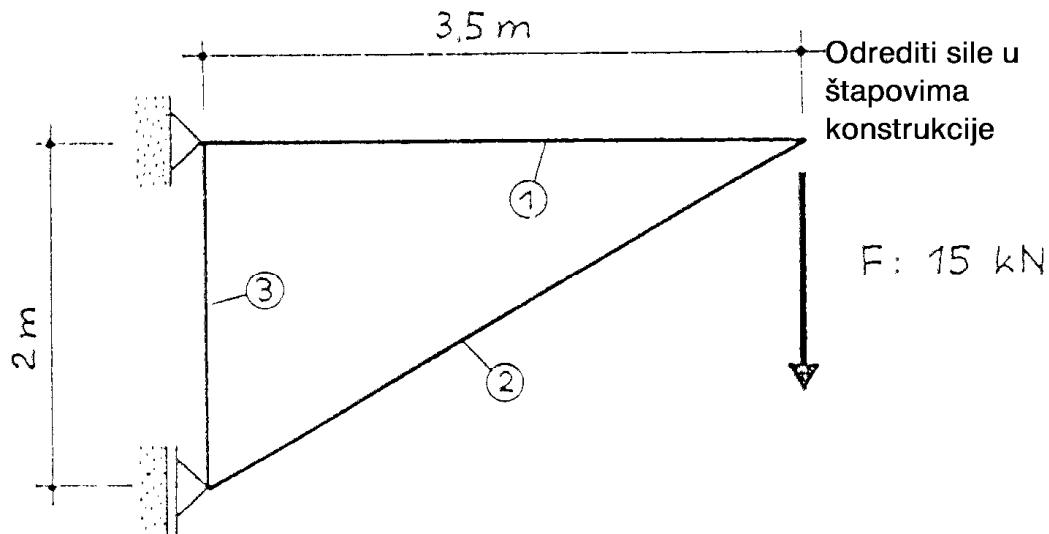
Primer razlaganja



$$V_A =$$

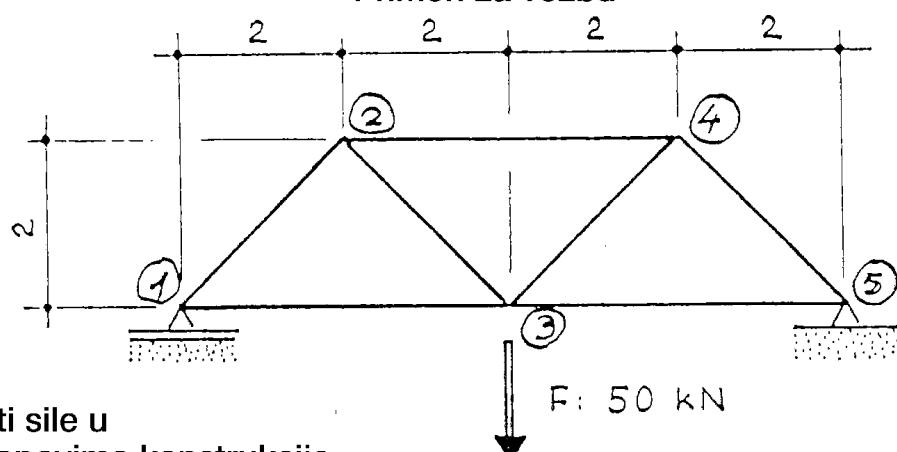
$$H_A =$$

$$M_A$$



①

Primeri za vežbu



Odrediti sile u
svim štapovima konstrukcije

$$N_{12} =$$

$$V_1 =$$

$$N_{13} =$$

$$N_{23} =$$

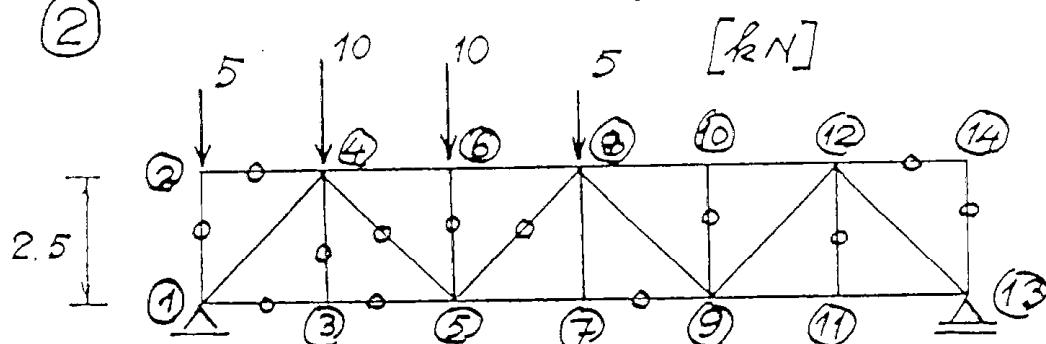
$$V_5 =$$

$$N_{35} =$$

$$N_{24} =$$

Odrediti sile u označenim štapovima

②



$$V_1 =$$

$$V_{13} =$$

$$6 \times 3 = 18 \text{ m}$$

Proveriti geometrijske krutosti i statičku određenos:

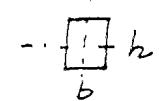
Formule za proračun napona

Zatezanje $\sigma_z = \frac{F}{A}$ $\sigma = \frac{N}{A}$

Pritisak $\sigma_{cl} = \frac{F}{A}$

Izvijanje $\sigma = \frac{N}{\varphi \cdot A}$; $\varphi \rightarrow \lambda = \frac{\ell_t}{i}$

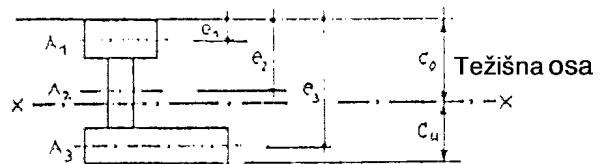
Savijanje $\sigma_b = \frac{M_b}{W_b}$
 $= \frac{M_b \cdot c}{I}$

$$W_b = \frac{b \cdot h^2}{6}$$


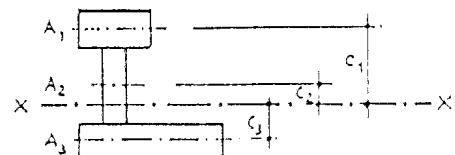
$$I_b = \frac{b \cdot h^3}{12}$$

c = Rastojanje težišta u ravni je potrebno za σ_b

$$c_0 = \frac{A_1 \cdot e_1 + A_2 \cdot e_2 + A_3 \cdot e_3}{A_1 + A_2 + A_3}$$



$$I_x = I_1 + A_1 \cdot c_1^2 + I_2 + A_2 \cdot c_2^2 + I_3 + A_3 \cdot c_3^2$$



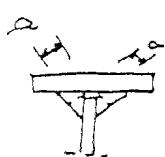
$$\tau_{II} = \frac{F_Q \cdot s}{I_x \cdot \Sigma a}$$

τ_{II} = Tangencijalni napon

F_Q = Poprečna sila

$$\tilde{\tau}_{II} = \frac{T \cdot s}{2a \cdot I_x}$$

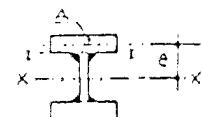
s = Statički moment
(Površina smicanja puta rastojanje težišta)



Σa = Debljina šava izložena smicanju



Presek



Smicanje $\tau_a = \frac{F}{A}$

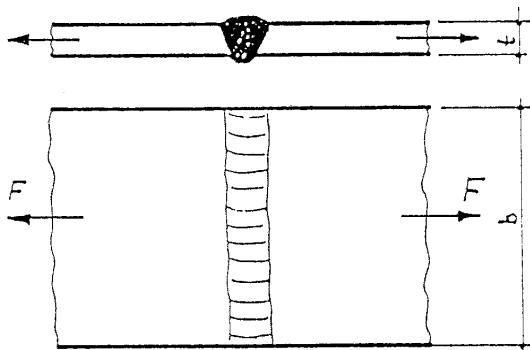
Uvijanje $\tilde{\tau}_t = \frac{M_t}{W_p}$
 $\approx \frac{M_t}{2A_R \cdot a}$

A_R = Od sredine profila obuhvaćena površina

a = Debljina profila/debljina šava u ravni, traži se za $\tilde{\tau}_t$



Primeri za proračun napona i dimenzija



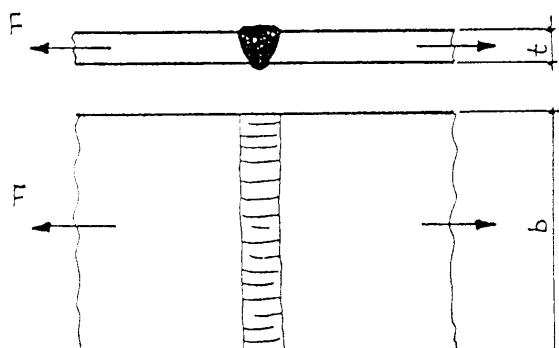
Spoj ploča (DIN 18 800)
 $F = 800 \text{ kN}$ (Opterećenje tipa H)
 Ploča 450×16 St 37-3
 Kvalitet šava nepoznat
 Napon ?

Uputstvo $\alpha = t$,

$$\sigma_{\text{II}} = \dots$$

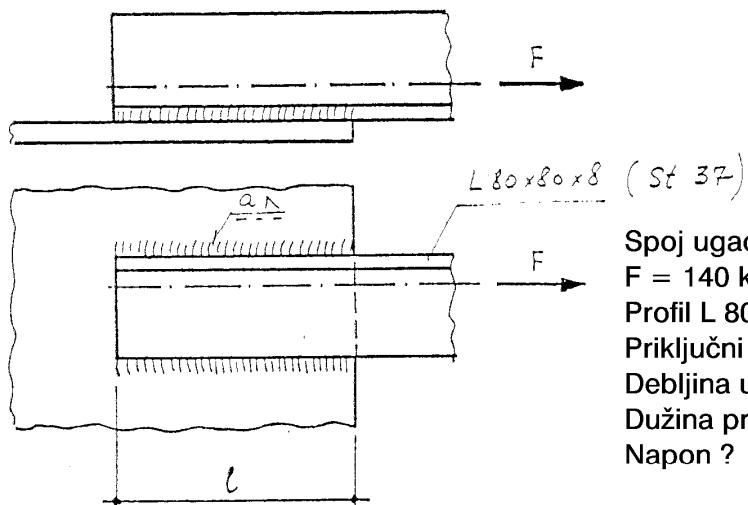
$$\tilde{\sigma}_{w,R,d} = \alpha_w f_y, k / \gamma_m$$

$$\tilde{\sigma}_{w,R,d} = 0,95 \cdot 24 / 1,1 \text{ kN/cm}^2$$



Spoj ploča (DIN 18 800)
 $F = 800 \text{ kN}$ (Opterećenje tipa HZ)
 Ploča debljine 16 mm; St 52-3
 Kvalitet šava nepoznat
 Širina ploče b ?

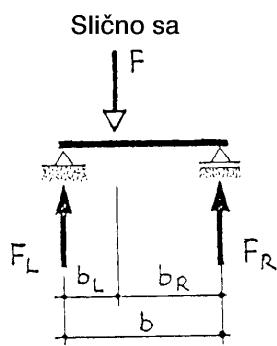
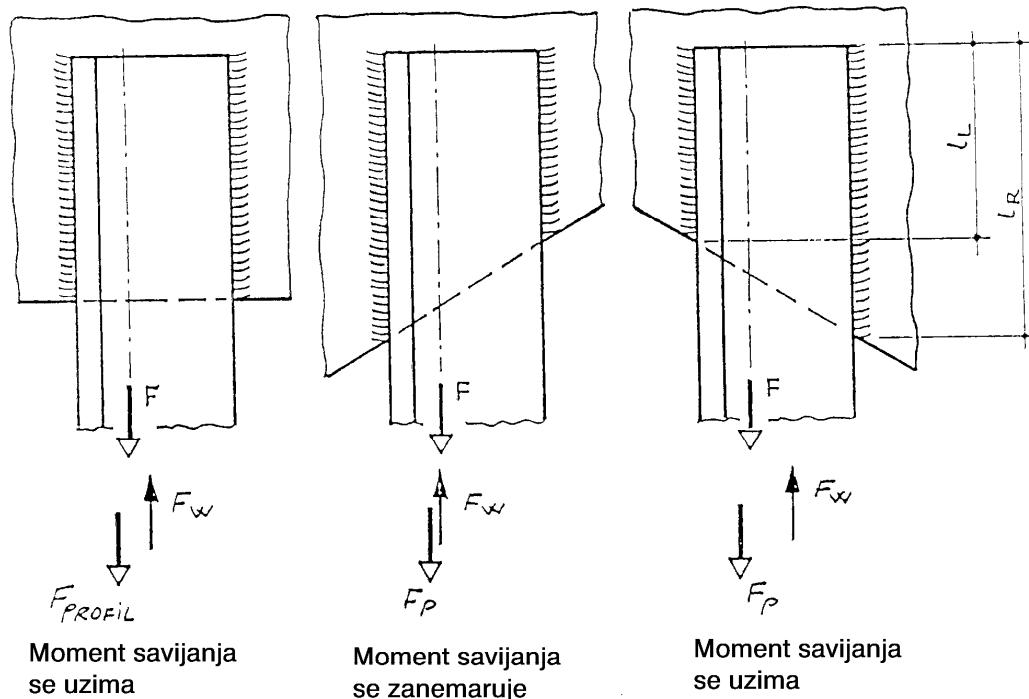
$$\tilde{\sigma}_{w,R,d} = 1,0 \cdot 24 / 1,1 \text{ kN/cm}^2$$



Spoj ugaonika (DIN 18 800)
 $F = 140 \text{ kN}$ (Opterećenje tipa H)
 Profil L 80X80X8; S235
 Priklučni lim 12 mm; S355
 Debljina ugaonog šava ?
 Dužina preklopa ?
 Napon ?

Uputstvo

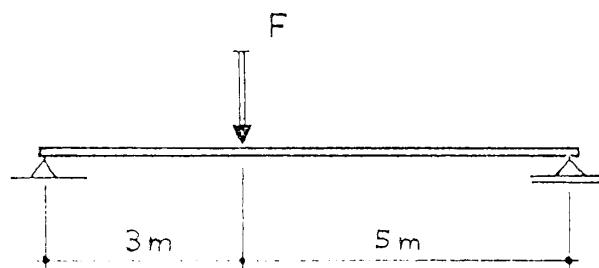
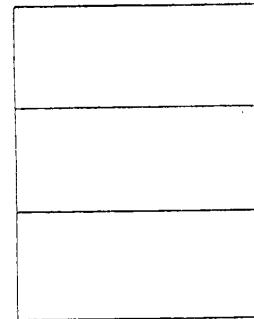
- $\tau_w = F / A_w$
- $\tau_w = 0,95 \cdot 24 / 1,1 \text{ kN/cm}^2$



$$F_L = \frac{F \cdot b_R}{b}$$

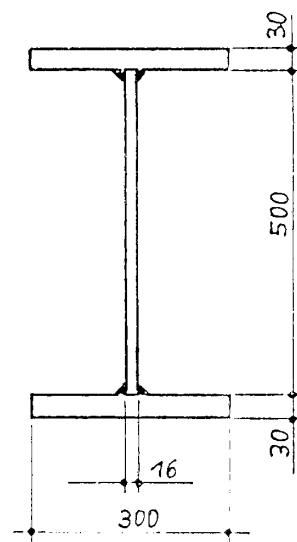
$$\alpha_{w,L} = \frac{F_L}{a_L \cdot l_L}$$

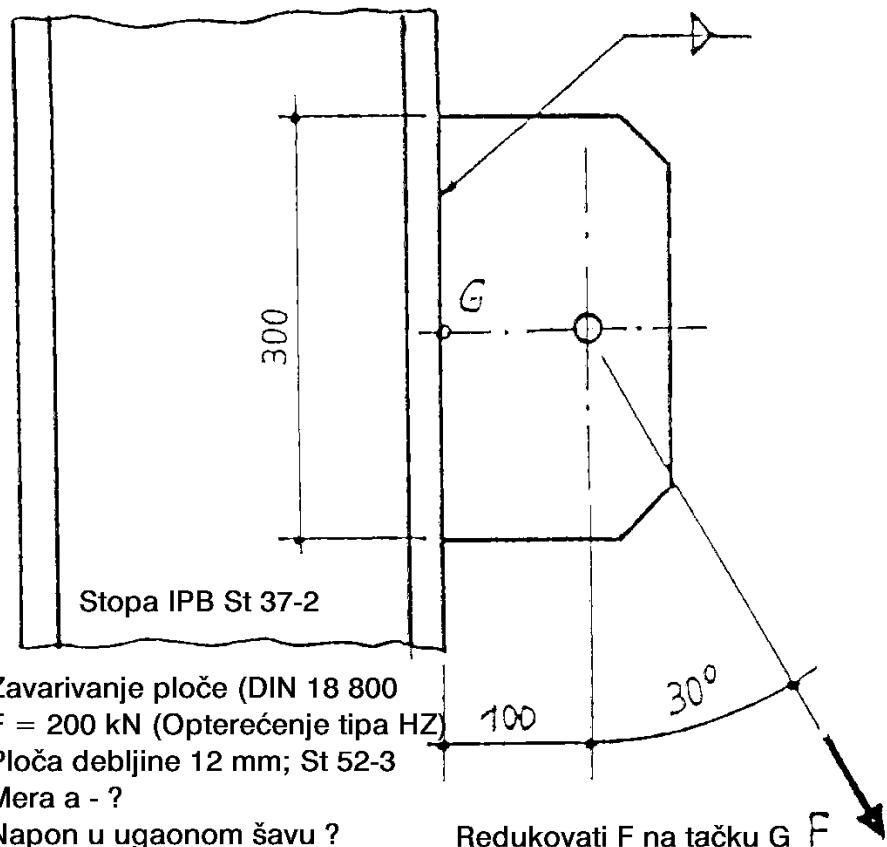
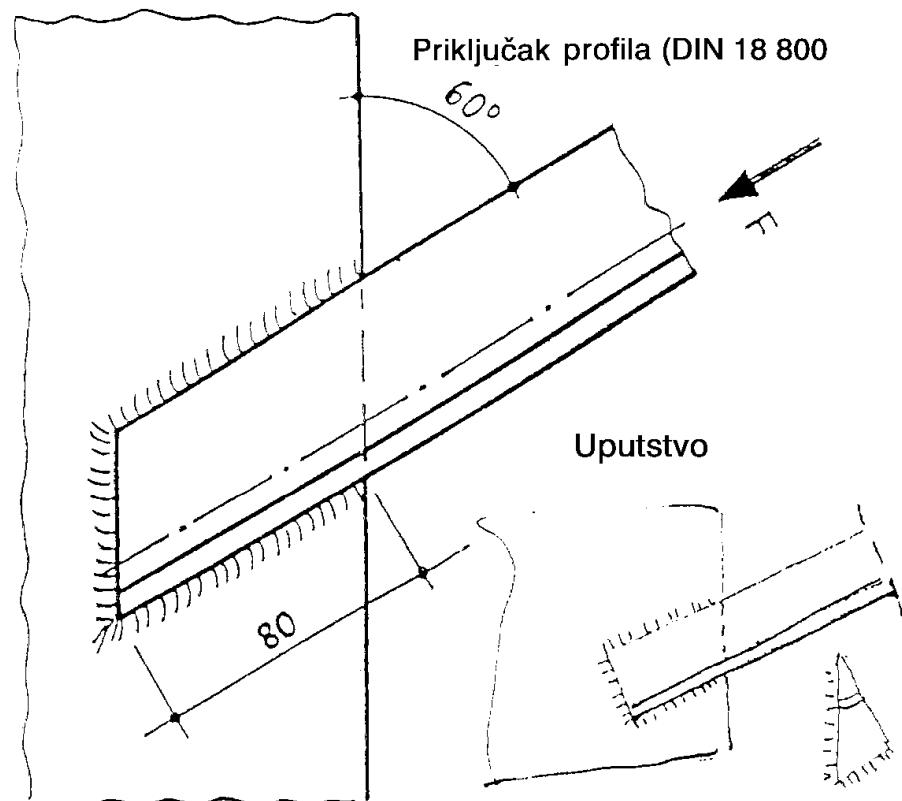
$$a_L = \alpha_{w,L} \cdot l_L$$

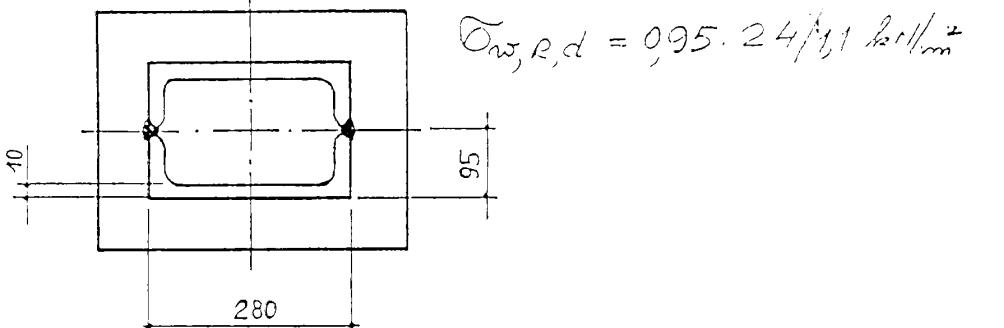
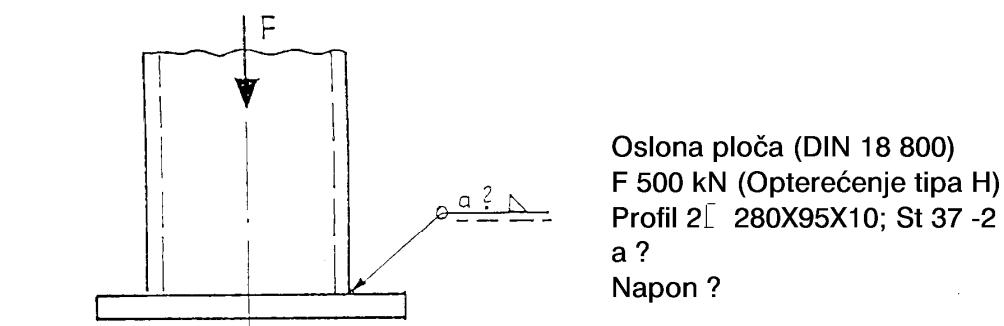
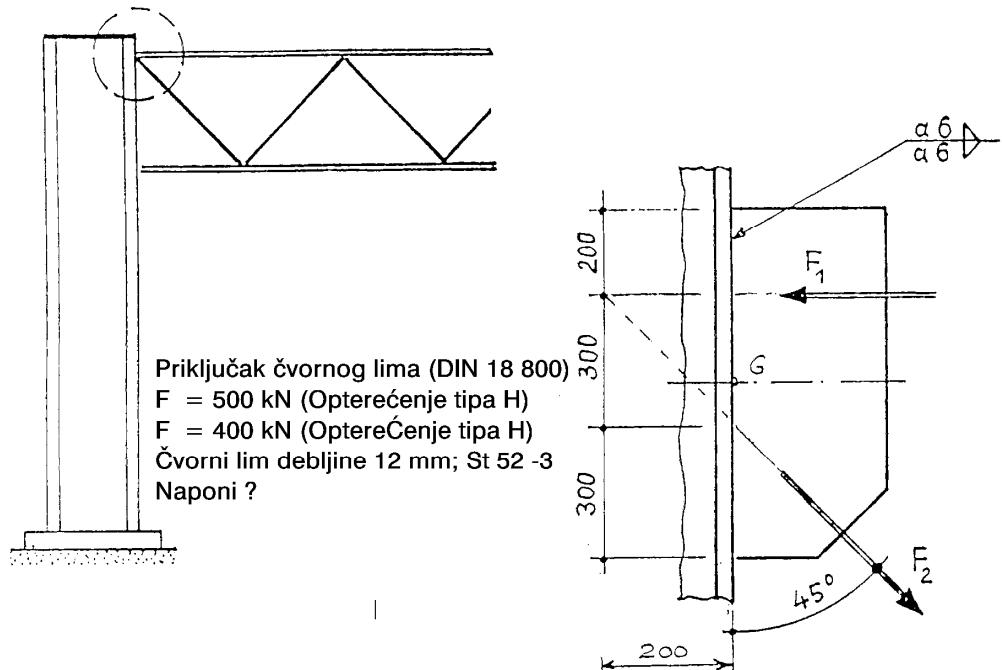


Zavarena greda (DIN 18 800)
 $F = 320 \text{ kN}$ (Opterećenje tipa H)
 Profil - gornji i donji pojasi 300X30; St 37-2
 Rebro 500Z16; St 37-2
 Mera a - minimalna
 Napon u nosaču i u ugaonom spoju ?

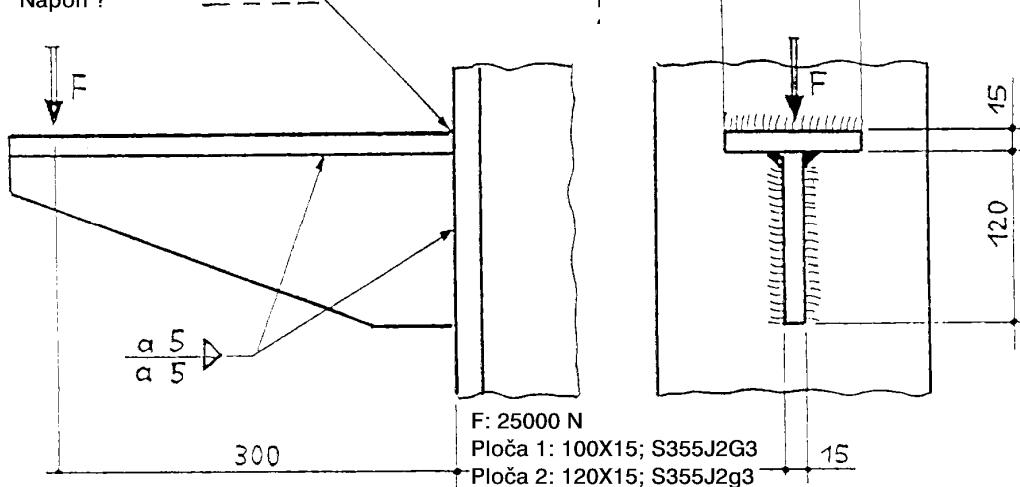
Uputstvo: nacrati dijagrame M, T.







Konzola
Napon ?



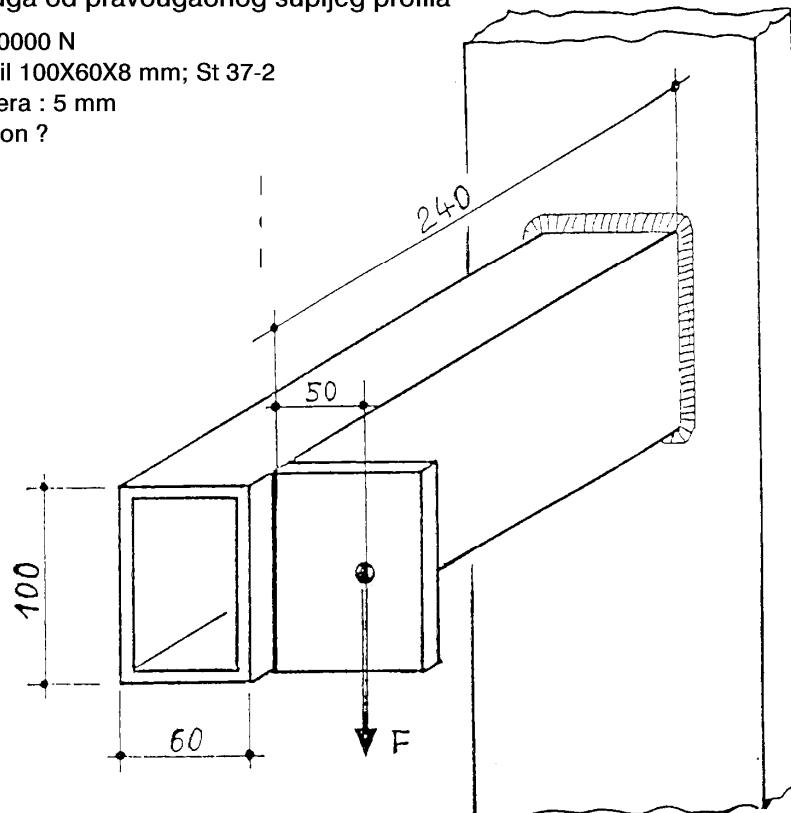
Poluga od pravougaonog šupljeg profila

F: 30000 N

Profil 100X60X8 mm; St 37-2

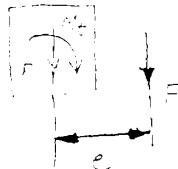
a mera : 5 mm

Napon ?



Uputstvo

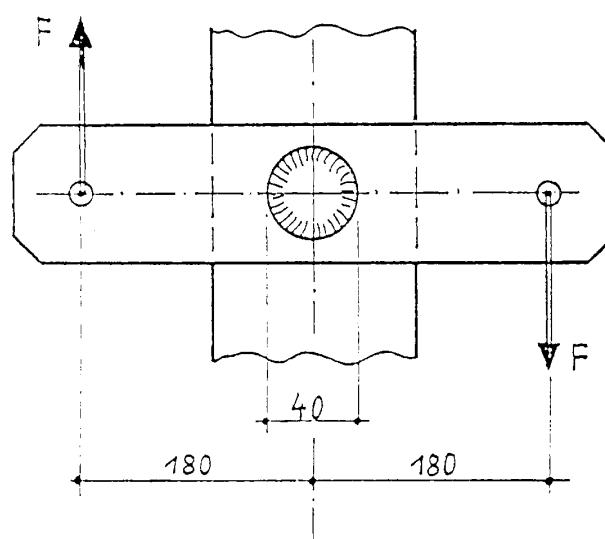
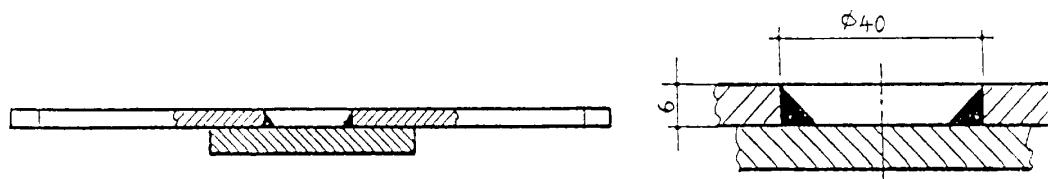
$$m_t \rightarrow \text{Bredt}$$



$$F \rightarrow$$



Formula
Bredt



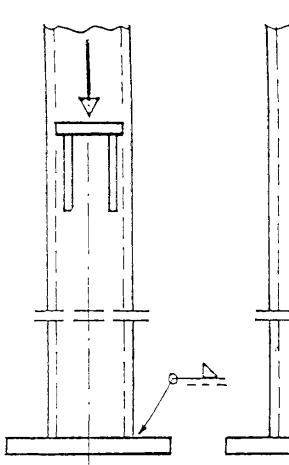
Zavarivanje čepa

F: 2,5 kN

Debljina lima 6 mm

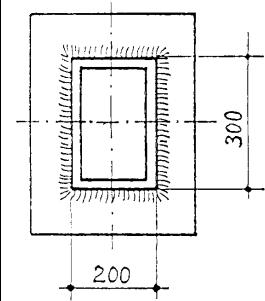
Precnik rupe 40 mm

Koristiti formulu Bretta

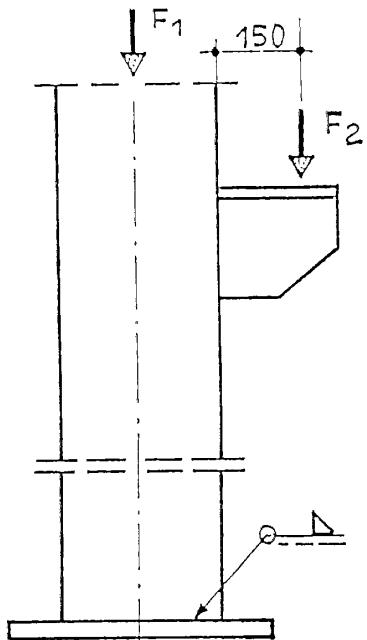
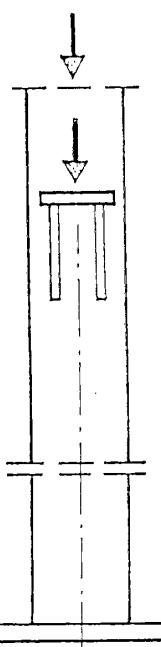
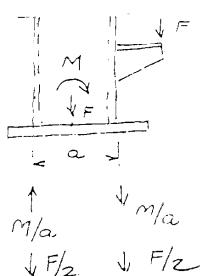


Oslona stopa (DIN 18 800)
F 160 kN
Potporni profil 300X200X12; S235
Osnovna ploča debljine 35 mm; S235
Debljin ugaonog šava a ?
Napon ?

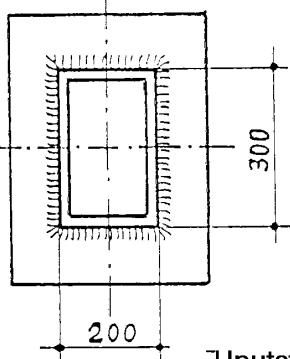
Upuststvo
Redukovati silu F
na težište spoja



$$\bar{\sigma}_{w,R,d} = 0,95 \cdot 24 / 31 \text{ kN/mm}^2$$



Oslona stopa
F1 200 kN
F2 100 kN
Potporni profil 300X200X12; S235
Osnovna ploča 35 mm; S235
Debljin ugaonog šava a ?
Napon ?



Upuststvo: vidi prethodni primer